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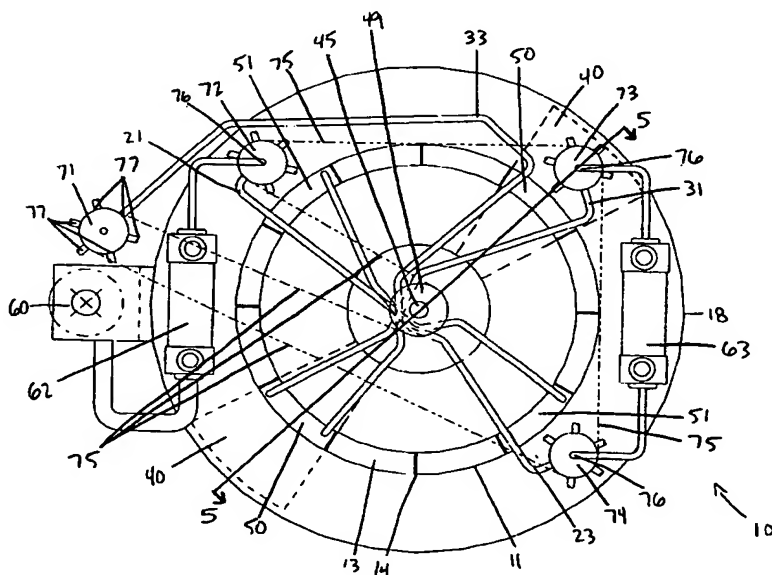
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(54) Title: ROTATING MAGNET MAGNETIC REFRIGERATOR



(57) Abstract: A magnetic refrigeration apparatus (10) has an annular container (11) including a plurality of magnetic regenerator compartments (13) containing magnetocaloric material (12), and a magnet (40) mounted for rotation around the annular container (11), whereby the motion of the magnet (40) produces a variation of magnetic field strength in the magnetic regenerator compartments, which in turn leads to a variation in temperature of the magnetocaloric material (12) in the magnetic regenerator compartments (13). Heat transfer fluid (17) is propelled by a pump (60), and directed to and from the regenerator compartments and hot and cold heat exchangers by valves. Each valve includes an axial port and a plurality of radial ports.



WO 03/050456 A1



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10/583912**IAP5 Rec'd PCT/PTO 20 JUN 2006****ROTATING MAGNET MAGNETIC REFRIGERATOR****FIELD OF THE INVENTION**

This invention relates generally to the field of magnetic refrigeration and to active magnetic regenerative refrigeration apparatus.

5

BACKGROUND OF THE INVENTION

Magnetic refrigeration, using magnetocaloric materials as a working element, promises to address important limitations associated with conventional refrigeration technologies which use gas compression and expansion. Magnetocaloric materials have the property that their
10 temperature changes when a magnetic field is applied. In the case of a material near a transition from a ferromagnetic state to a paramagnetic state, the material will warm when magnetized and cool when demagnetized. Magnetic refrigeration can avoid the use of volatile fluids, such as chlorofluorocarbons (CFC's), that may harm the environment.
15 Magnetic refrigeration can be more energy efficient than conventional refrigeration technologies. Magnetic refrigeration can also produce very low temperatures, which can enable, for example, cost-effective production of liquid hydrogen for use as an alternative fuel for transportation and other applications. Thus, there has long been
20 motivation to find an effective apparatus for magnetic refrigeration.

Many magnetic refrigerators use active magnetic regeneration as an operating principle. The term active means that a magnetic field is applied to a magnetocaloric material and then removed. A regenerator is a thermal device that transfers heat into a heat transfer medium during
25 one stage of a regenerative cycle, and then transfers heat out of that heat transfer medium during an opposite phase of the regenerative cycle.

Active magnetic regeneration refers to a regenerator which applies a time-varying magnetic field and reciprocating flow of a heat transfer medium to an elongated container of magnetocaloric materials, to produce a temperature gradient along the container of magnetocaloric materials and to enable heat transfer into and out of the heat transfer medium. Active magnetic regeneration may be used in a magnetic refrigerator, to provide cooling, or in a heat pump, to provide heating.

In a typical active magnetic regenerator device, a bed of magnetocaloric material which is porous to a heat transfer fluid is connected to two heat exchangers, with mechanisms provided for magnetizing and demagnetizing the bed, and for effecting reciprocating fluid flow through the bed of magnetocaloric material from one heat exchanger to the other. A typical active magnetic regenerator device usually performs four basic operations: (1) bed magnetization, which increases the temperature of the magneto-caloric material in the bed by the magnetocaloric effect; (2) fluid transfer in the cold side to hot side direction, with warmed fluid flowing out of the bed into a hot side heat exchanger, where heat can be released; (3) bed demagnetization, which reduces the temperature of the magnetocaloric material in the bed by the magnetocaloric effect; and (4) fluid transfer in the hot side to cold side direction, with cooled fluid flowing out of the bed and into a cold side heat exchanger, where heat can be absorbed.

SUMMARY OF THE INVENTION

In accordance with the present invention, a magnetic refrigeration apparatus has a magnetic regenerator bed containing magnetocaloric material, a magnet, and a means for moving the magnet in a path adjacent to the magnetic regenerator bed, whereby the motion of the magnet produces a variation of magnetic field strength in the magnetic

regenerator bed, which in turn leads to a variation in temperature of the magnetocaloric material.

In one aspect of the invention, the magnetic regenerator bed is comprised of a plurality of compartments arranged in a ring, and a magnet
5 is attached to a rotating assembly which moves the magnet in a path around the ring.

In another aspect of the invention, a method of transferring heat comprises rotating a magnet around an annular container which includes a magnetic regenerator compartment containing magnetocaloric material.
10 The rotation of the magnet produces a cyclic variation in magnetic field strength in the magnetic regenerator compartment. The variation in magnetic field strength causes a cyclic variation in temperature of the magnetocaloric material in the magnetic regenerator compartment.

In another aspect of the invention, a heat transfer apparatus
15 comprises an annular container which includes a plurality of magnetic regenerator compartments, a magnet mounted for rotation about the central axis of the annular container, a heat exchanger, and a valve. Each magnetic regenerator compartment has a hot side and a cold side, and each magnetic regenerator compartment contains magnetocaloric material
20 that allows the flow of heat transfer fluid through such magnetocaloric material. The valve has an axial port and a plurality of radial ports. The axial port of the valve is connected by a pipe to the hot heat exchanger, and each radial port of the valve is connected by a pipe to the hot side of a magnetic regenerator compartment. The rotation of the magnet
25 produces a cyclic variation in magnetic field strength in the magnetic regenerator compartments. The variation in magnetic field strength causes a cyclic variation in temperature of the magnetocaloric material in the magnetic regenerator compartments. The valve is used to cause the

heat transfer fluid to flow from the magnetic regenerator compartments to and from the heat exchanger at the appropriate time to exploit the cyclic variation of temperature of a magnetocaloric material for heat transfer.

In another aspect of the invention, a heat transfer apparatus
5 includes an annular arrangement of a plurality of magnetic regenerator compartments containing magnetocaloric material, a magnet mounted for rotation about the central axis of the annular arrangement of magnetic regenerator compartments, and a valve having an axial port, a rotating inner assembly, and a plurality of radial ports, wherein the rotating inner
10 assembly rotates synchronously with the rotation of the magnet to connect heat transfer fluid flow between the axial port and one or more of the radial ports.

A rotating magnet magnetic refrigerator according to such a preferred embodiment of the invention has several desirable features.
15 Work input to the device is via circular motion that may be at constant speed. Forces are well balanced, so that the net drive force is mainly that necessary to drive the refrigeration process, and this force is nearly constant. Reciprocating flow occurs to the magnetocaloric material, allowing regenerative cycles to be performed, and yet dead volume
20 effects in the heat exchangers or between the magnetocaloric material and the heat exchangers are minimized. Finally, the seals used in the valve can be of simple design, are exposed to minimal wear, and generate minimal friction.

Further objects, features and advantages of the invention will be
25 apparent from the following detailed description when taken in conjunction with the accompanying drawings.

BRIEF DESCRIPTION OF THE DRAWINGS

In the drawings:

Fig. 1 is a plan view of a rotating magnet magnetic refrigerator in accordance with the invention.

5 Fig. 2 is a plan view of an exemplary compartment containing magnetocaloric material.

Fig. 3 is a plan view of the rotating magnet magnetic refrigerator of Fig. 1, with the heat transfer fluid components removed.

10 Fig. 4 is a plan view of the heat transfer fluid components in the rotating magnet magnetic refrigerator of Fig. 1.

Fig. 5 is a cross-sectional view of the magnetic refrigerator taken generally along the lines 5-5 of Fig. 1.

Fig. 6 is a cross-sectional view of an exemplary magnet for use in the magnetic refrigerator of Fig. 1.

15 Fig. 7 is a cross-sectional view of an exemplary valve for use in the magnetic refrigerator of Fig. 1.

DETAILED DESCRIPTION OF THE INVENTION

A preferred embodiment of a rotating magnet refrigerator according to the invention, indicated generally at 10, uses a stationary annular (ring
20 shaped) container 11 of magnetocaloric material 12 separated into a number of compartments 13 (12 compartments are shown in Fig. 1) by radial boundaries 14, as illustrated in Figs. 1-3. These radial boundaries 14 impede the flow of fluid and heat. Each compartment 13 has a cold side 15 and a hot side 16, and the magnetocaloric material 12 therein is

porous to fluid flow, allowing heat transfer fluid 17 to be made to flow alternately from the cold side 15 to the hot side 16, or from the hot side 16 to the cold side 15.

As best shown in Figs. 2 and 4, each compartment 13 has a pair of
5 fluid access ports and associated pipes at its cold side 15 including a cold side inlet pipe 21 connected to a cold side inlet port 22 and a cold side outlet port 23 connected to a cold side outlet pipe 24, and a pair of fluid access ports and associated pipes at its hot side 16, composed of a hot side inlet pipe 31 and hot side inlet port 32 and a hot side outlet port 34
10 and hot side outlet pipe 33.

One or more magnets 40 are mounted for circular motion to allow them to be circularly driven around the stationary annular container 11 of magnetocaloric material 12. As best shown in Fig. 5, each magnet 40 may be mounted on a rotating magnet mount 43, which may be driven by
15 a motor 44. As best shown in Fig. 6, each magnet 40 may have opposing faces 53 which focus the magnetic flux through a compartment 13 containing magnetocaloric material 12. As best shown in Figs. 5 and 6, the magnet 40 is a permanent magnet which may be comprised of one or more permanently magnetized sections 41 and one or more iron
20 sections 42.

As best shown in Fig. 3, the magnet design is such that the flux emerging from the magnets 40 through faces 53 is concentrated in one or more compartments 13 that are in regions 50 that are nearest the magnets 40, while almost no flux enters those compartments 13 that are
25 in regions 51 that are far from the magnets 40. An intermediate level of flux may enter compartments 13 that are in regions 52 that are at an intermediate distance from the magnets 40. The motion of the magnets 40 thus produces a cyclic variation of magnetic field strength at each

compartment 13, which in turn leads to cyclic variation in temperature of the magnetocaloric material 12 via the magnetocaloric effect. At a given time, those compartments 13 that are in regions 50 will be at a relatively high magnetic field, those compartments 13 that are in regions 51 will be
5 at a relatively low field, and those compartments 13 that are in regions 52 will be at a field of intermediate strength.

As best shown in Fig. 4, a heat transfer fluid pump 60, which may be run at constant speed, is connected to a fluid flow circuit composed of a heat transfer fluid 17 suffusing the circuit, a hot heat exchanger 62, a
10 cold heat exchanger 63, a number of valves 71-74, the compartments 13, and connecting piping and ports. Only one-sixth of the pipes associated with the beds are shown in Figs. 1 and 4.

As best shown in Fig. 5, the motor 44 may include a motor shaft 45. A pump drive pulley 46 may be attached to the motor shaft 45, and
15 a pump drive belt 61 may be used to drive the heat transfer fluid pump 60. A speed reducer 47 which includes a speed reducer shaft 48 may also be attached to the motor shaft 45. As best shown in Figs. 1 and 5, a valve drive pulley 49 may be attached to the speed reducer shaft 48, and a valve drive belt 75 may be used to drive the valves 71-74.

20 As best shown in Figs. 1 and 4, at the time when the magnets 40 are sitting over the compartments 13 that are located in regions 50, the valves 71-74 are set into positions such that heat transfer fluid 17 entering the valve 73 through axial port 76 from pipe 83 is directed by the valve 73 through a radial port 77 to cold side inlet pipes 21 and cold
25 side inlet ports 22 to the compartments 13 that are in regions 50 at high magnetic field, where the heat transfer fluid 17 is warmed by the magnetocaloric material 12, then through hot side outlet ports 34 and hot side outlet pipes 33 to a radial port 77 on valve 71, thence through the

axial port 76 to pipe 81 to the fluid pump 60, then through pipe 85 to the hot heat exchanger 62, where heat is given off to the environment.

As best shown in Figs. 1 and 6, the heat transfer fluid leaves the hot heat exchanger 62 and then passes through the pipe 82 to the axial
5 port 76 on the valve 72, where heat transfer fluid 17 is directed through a radial port 77 to hot side inlet pipes 31 and hot side inlet ports 32 to the compartments 13 that are in regions 51 at low magnetic field, where the heat transfer fluid 17 is cooled, and then through cold side outlet
ports 24 and cold side outlet pipes 23 to a radial port 77 on valve 74,
10 thence through the axial port 76 and through pipe 84 to the cold heat exchanger 63, where the thermal load is cooled.

As the magnet(s) 40 are moved around the stationary annular container 11, different compartments 13 are exposed to high and low magnetic field, and the setting of the valves 71-74 and thus the flow in
15 the piping and in the compartments 13, are changed accordingly. The valves 71-74 are set such that the flow of heat transfer fluid 17 in the hot and cold heat exchangers 62-63, and in each pipe 81-85 between the fluid pump 60, valves 71-74 and the heat exchangers 62-63 is in a single direction. Moreover, the flow of heat transfer fluid 17 in the piping
20 between the compartments 13 and the heat exchangers 62-63 is set such that the flow in each pipe is either in a unique direction, or is zero.

By correct setting of the valves 71-74 it is thus possible to avoid the effects of dead volume in the heat exchangers 62-63 or the piping to the heat exchangers 62-63 by ensuring unidirectional flow everywhere
25 except inside the compartments 13, where correctly timed reversing flow occurs. The only seals 78 that are exposed to moving surfaces, and that thus possibly generate frictional heating, are in the valves 71-74 and

perhaps the pump 60. These seals are compact and are exposed to relatively low surface velocities.

There are various possibilities with regard to alternative embodiments of a magnetic refrigeration apparatus according to the
5 invention.

In the above described embodiment, two magnets 40 are used, and flow from the cold heat exchanger 63 is directed to a single pair of compartments 13 in the regions 50 at high magnetic field through a single port of the valve 73, but this is not required. The magnets 40 may cast a
10 high magnetic field over more than one pair of compartments 13 at a given time, in which case it is advantageous for flow from the cold heat exchanger 63 to be directed simultaneously to more than one pair of compartments 13. This may be done with the same piping system as described above by changing the valves 71-74 such that flow occurs
15 simultaneously through multiple radial ports 77. The valves 71-74 may also be constructed so that the flow to a given radial port 77 turns on gradually, which can be made to occur in synchrony with a gradual increase or decrease in magnetic field at the corresponding compartment 13. A different number of magnets 40 or a different number of
20 compartments 13 can be handled by a similar flow system to that described above, but with a different arrangement of valves and pipes. The valves 71-74 may be multiple position valves, two position valves, or a combination of multiple position and check valves.

Although in a preferred embodiment, an apparatus according to the
25 invention may be used as a magnetic refrigerator, other applications of the apparatus are possible. For example, the apparatus may operate as a heat pump to provide heating by connecting the hot heat exchanger 62 to the body to be heated, and by connecting the cold heat exchanger 63 to

the environment. Similarly, the apparatus may be used in an air conditioner to provide residential cooling, or in any application which utilizes heat transfer to provide a useful result. Additional flow ports, heat exchangers or pumps may also be used.

5 Although in a preferred embodiment the compartments 13 with ports as discussed above provide fluid flow in a circumferential direction, as best shown in Figs. 1, 2, and 4, this is not required. Alternatively, the stationary annular container 11 may have compartments 13 which are constructed for radial or axial flow. The magnetocaloric material 12 must
10 be porous to flow, but may be in the form of particles, or thin sheets, or other high surface area geometries, which may be packed in simple physical contact, or bonded together. If unbonded particles are used, they may be prevented from escaping the compartment by use of screens or finely perforated sheets covering the inlet and exit ports.

15 Although in a preferred embodiment a liquid is used as a heat transfer fluid, other media for heat transfer may be used. For example, a gas could be used as a heat transfer medium, alone or in combination with a liquid.

 Although in a preferred embodiment a stationary annular (ring
20 shaped) container 11 of magnetocaloric material 12 separated into a number of compartments 13 is used, other arrangements of magnetocaloric material 12 could be used. For example, the container 11 of magnetocaloric material 12 may be formed as a disk having no central hole. There could be a greater or lesser number of compartments, there
25 may be gaps in the container 11 between the compartments 13 containing magnetocaloric material 12, or there may be portions of the container 11 which do not contain magnetocaloric material 12. The

container 11 may be comprised of a plurality of segments, or form a polygon which approximates an annular shape.

Although in a preferred embodiment, two magnets comprised of multiple magnet sections and multiple iron sections are used, other
5 magnet arrangements could be used. For example, there could be a greater or lesser number of magnet sections, or a greater or lesser number of iron sections, or a greater or lesser number of magnets.

Although in a preferred embodiment, the magnetocaloric material is near a ferromagnetic to paramagnetic transition, in which case the
10 material heats when magnetized and cools when demagnetized, other types of magnetocaloric materials may be used that cool when magnetized and heat when demagnetized. In the latter case, the fluid flow directions in the magnetized and demagnetized compartments would be in the reverse sense to that described above.

15 It is understood that the invention is not confined to the particular embodiments set forth herein as illustrative, but embraces all such forms thereof as come within the scope of the following claims.

CLAIMS

WHAT IS CLAIMED IS:

- 1 1. A method of transferring heat comprising the steps of:
 - 2 (a) providing an annular container having a center and
 - 3 including a magnetic regenerator compartment arranged at a selected
 - 4 radius from the center, wherein the magnetic regenerator compartment
 - 5 has a hot side and a cold side;
 - 6 (b) providing magnetocaloric material located in the
 - 7 magnetic regenerator compartment, wherein the magnetocaloric material
 - 8 is configured to allow fluid flow through such magnetocaloric material;
 - 9 (c) providing a rotatable magnet assembly which includes
 - 10 a magnet located adjacent to a portion of the magnetic regenerator
 - 11 compartment;
 - 12 (d) rotating the magnet assembly to produce a cyclic
 - 13 variation in magnetic field strength in the magnetic regenerator
 - 14 compartment as the magnet assembly rotates;
 - 15 (e) providing a heat transfer fluid; and
 - 16 (f) passing the heat transfer fluid through the magnetic
 - 17 regenerator compartment.
- 1 2. The method of claim 1 wherein the step of passing the
- 2 heat transfer fluid through the magnetic regenerator compartment
- 3 includes the steps of:
 - 4 (a) passing the heat transfer fluid through the magnetic
 - 5 regenerator compartment from the hot side to the cold side of the
 - 6 magnetic regenerator compartment when the magnetic field strength in
 - 7 the magnetic regenerator compartment is relatively low; and

8 (b) passing the heat transfer fluid through the magnetic
9 regenerator compartment from the cold side to the hot side of the
10 magnetic regenerator compartment when the magnetic field strength in
11 the magnetic regenerator compartment is relatively high.

1 3. The method of claim 1 wherein the cold side of each
2 magnetic regenerator compartment is adjacent to the cold side of an
3 adjacent magnetic regenerator compartment and the hot side of each
4 magnetic regenerator compartment is adjacent to the hot side of an
5 adjacent magnetic regenerator compartment.

1 4. The method of Claim 1 wherein the step of passing
2 the heat transfer fluid through the magnetic regenerator compartment
3 includes passing the heat transfer fluid through the magnetic regenerator
4 compartment from the hot side to the cold side of the magnetic
5 regenerator compartment when the magnetic field strength in the
6 magnetic regenerator compartment is relatively low, and further
7 comprising the steps of:

8 (a) providing a hot heat exchanger; and

9 (b) passing the heat transfer fluid through the hot heat
10 exchanger after passing the heat transfer fluid through the magnetic
11 regenerator compartment from the cold side to the hot side of the
12 magnetic regenerator compartment.

1 5. The method of Claim 1 wherein the step of passing
2 the heat transfer fluid through the magnetic regenerator compartment
3 includes passing the heat transfer fluid through the magnetic regenerator
4 compartment from the cold side to the hot side of the magnetic
5 regenerator compartment when the magnetic field strength in the

6 magnetic regenerator compartment is relatively high, and further
7 comprising the steps of:

- 8 (a) providing a cold heat exchanger; and
9 (b) passing the heat transfer fluid through the cold heat
10 exchanger after passing the heat transfer fluid through the magnetic
11 regenerator compartment from the hot side to the cold side of the
12 magnetic regenerator compartment.

1 6. The method of Claim 1 wherein the step of passing
2 the heat transfer fluid through the magnetic regenerator compartment
3 includes passing the heat transfer fluid through the magnetic regenerator
4 compartment from the cold side to the hot side of the magnetic
5 regenerator compartment when the magnet is adjacent to the magnetic
6 regenerator compartment.

1 7. The method of Claim 1 wherein the step of passing
2 the heat transfer fluid through the magnetic regenerator compartment
3 includes passing the heat transfer fluid through the magnetic regenerator
4 compartment from the hot side to the cold side of the magnetic
5 regenerator compartment when the magnet is not adjacent to the
6 magnetic regenerator compartment.

1 8. A heat transfer apparatus comprising:
2 (a) an annular container having a central axis, and which
3 includes a plurality of magnetic regenerator compartments, each magnetic
4 regenerator compartment containing magnetocaloric material that allows
5 the flow of heat transfer fluid through such magnetocaloric material, and
6 each magnetic regenerator compartment having a hot side and a cold
7 side;

8 (b) a magnet mounted for rotation about the central axis
9 of the annular container, the magnet providing a magnetic field that at a
10 first rotational position passes a relatively high magnetic field strength
11 through a first magnetic regenerator compartment in the plurality of
12 magnetic regenerator compartments and passes a relatively low magnetic
13 field strength through a second magnetic regenerator compartment in the
14 plurality of magnetic regenerator compartments, wherein at a second
15 rotational position the magnet passes a relatively low magnetic field
16 strength through the first magnetic regenerator compartment and passes
17 a relatively high magnetic field strength through the second magnetic
18 regenerator compartment;

19 (c) a hot heat exchanger; and

20 (d) a valve having an axial port, a first radial port and a
21 second radial port, the axial port of the valve connected by a first pipe to
22 the hot heat exchanger, the first radial port of the valve connected by a
23 second pipe to the hot side of the second magnetic regenerator
24 compartment.

1 9. The apparatus of Claim 8 wherein the heat transfer
2 fluid flows circumferentially through the second magnetic regenerator
3 compartment.

1 10. The apparatus of Claim 8 wherein there are an even
2 number of magnetic regenerator compartments in the plurality of
3 magnetic regenerator compartments.

1 11. The apparatus of Claim 10 wherein the cold side of
2 each magnetic regenerator compartment is adjacent to the cold side of an
3 adjacent magnetic regenerator compartment, and the hot side of each

4 magnetic regenerator compartment is adjacent to the hot side of an
5 adjacent magnetic regenerator compartment.

1 12. The apparatus of Claim 11 wherein each magnetic
2 regenerator compartment further comprises a cold side input port, and
3 wherein the cold side input port of each magnetic regenerator
4 compartment is adjacent to the cold side input port of an adjacent
5 magnetic regenerator compartment, and the cold side input port of each
6 magnetic regenerator compartment is open to the adjacent cold side input
7 port of an adjacent magnetic regenerator compartment for fluid flow.

1 13. The apparatus of Claim 11 wherein each magnetic
2 regenerator compartment further comprises a cold side output port, and
3 wherein the cold side output port of each magnetic regenerator
4 compartment is adjacent to the cold side output port of an adjacent
5 magnetic regenerator compartment, and the cold side output port of each
6 magnetic regenerator compartment is open to the adjacent cold side
7 output port of an adjacent magnetic regenerator compartment for fluid
8 flow.

1 14. The apparatus of Claim 11 wherein each magnetic
2 regenerator compartment further comprises a hot side input port, and
3 wherein the hot side input port of each magnetic regenerator
4 compartment is adjacent to the hot side input port of an adjacent
5 magnetic regenerator compartment, and the hot side input port of each
6 magnetic regenerator compartment is open to the hot side input port of an
7 adjacent magnetic regenerator compartment for fluid flow.

1 15. The apparatus of Claim 11 wherein each magnetic
2 regenerator compartment further comprises a hot side output port, and
3 wherein the hot side output port of each magnetic regenerator

4 compartment is adjacent to the hot side output port of an adjacent
5 magnetic regenerator compartment, and the hot side output port of each
6 magnetic regenerator compartment is open to the hot side output port of
7 an adjacent magnetic regenerator compartment for fluid flow.

1 16. A heat transfer apparatus comprising:

2 (a) an annular container having a central axis, and which
3 includes a plurality of magnetic regenerator compartments, each magnetic
4 regenerator compartment containing magnetocaloric material that allows
5 the flow of heat transfer fluid through such magnetocaloric material, and
6 each magnetic regenerator compartment having a hot side and a cold
7 side;

8 (b) a magnet mounted for rotation about the central axis
9 of the annular container, the magnet providing a magnetic field that at a
10 first rotational position passes a relatively high magnetic field strength
11 through a first magnetic regenerator compartment in the plurality of
12 magnetic regenerator compartments and passes a relatively low magnetic
13 field strength through a second magnetic regenerator compartment in the
14 plurality of magnetic regenerator compartments, wherein at a second
15 rotational position the magnet passes a relatively low magnetic field
16 strength through the first magnetic regenerator compartment and passes
17 a relatively high magnetic field strength through the second magnetic
18 regenerator compartment;

19 (c) a cold heat exchanger; and

20 (d) a valve having an axial port, a first radial port and a
21 second radial port, the axial port of the valve connected by a first pipe to
22 the cold heat exchanger, the first radial port of the valve connected by a
23 second pipe to the cold side of the first magnetic regenerator
24 compartment.

1 17. The apparatus of Claim 16 wherein the heat transfer
2 fluid flows circumferentially through the first magnetic regenerator
3 compartment.

1 18. The apparatus of Claim 16 wherein there are an even
2 number of magnetic regenerator compartments in the plurality of
3 magnetic regenerator compartments.

1 19. The apparatus of Claim 16 wherein the cold side of
2 each magnetic regenerator compartment is adjacent to the cold side of an
3 adjacent magnetic regenerator compartment, and the hot side of each
4 magnetic regenerator compartment is adjacent to the hot side of an
5 adjacent magnetic regenerator compartment.

1 20. The apparatus of Claim 19 wherein each magnetic
2 regenerator compartment further comprises a cold side input port, and
3 wherein the cold side input port of each magnetic regenerator
4 compartment is adjacent to the cold side input port of an adjacent
5 magnetic regenerator compartment, and the cold side input port of each
6 magnetic regenerator compartment is open to the adjacent cold side input
7 port of an adjacent magnetic regenerator compartment for fluid flow.

1 21. The apparatus of Claim 19 wherein each magnetic
2 regenerator compartment further comprises a cold side output port, and
3 wherein the cold side output port of each magnetic regenerator
4 compartment is adjacent to the cold side output port of an adjacent
5 magnetic regenerator compartment, and the cold side output port of each
6 magnetic regenerator compartment is open to the adjacent cold side
7 output port of an adjacent magnetic regenerator compartment for fluid
8 flow.

1 22. The apparatus of Claim 19 wherein each magnetic
2 regenerator compartment further comprises a hot side input port, and
3 wherein the hot side input port of each magnetic regenerator
4 compartment is adjacent to the hot side input port of an adjacent
5 magnetic regenerator compartment, and the hot side input port of each
6 magnetic regenerator compartment is open to the adjacent hot side input
7 port of an adjacent magnetic regenerator compartment for fluid flow.

1 23. The apparatus of Claim 19 wherein each magnetic
2 regenerator compartment further comprises a hot side output port, and
3 wherein the hot side output port of each magnetic regenerator
4 compartment is adjacent to the hot side output port of an adjacent
5 magnetic regenerator compartment, and the hot side output port of each
6 magnetic regenerator compartment is open to the adjacent hot side
7 output port of an adjacent magnetic regenerator compartment for fluid
8 flow.

1 24. A heat transfer apparatus comprising:

- 2 (a) an annular container having a central axis, and which
3 includes a plurality of magnetic regenerator compartments, each magnetic
4 regenerator compartment containing magnetocaloric material that allows
5 the flow of heat transfer fluid through such magnetocaloric material, and
6 each magnetic regenerator compartment having a hot side and a cold
7 side;
- 8 (b) a magnet mounted for rotation about the central axis
9 of the annular container, the magnet providing a magnetic field that at a
10 first rotational position passes a relatively high magnetic field strength
11 through a first magnetic regenerator compartment in the plurality of
12 magnetic regenerator compartments and passes a relatively low magnetic
13 field strength through a second magnetic regenerator compartment in the

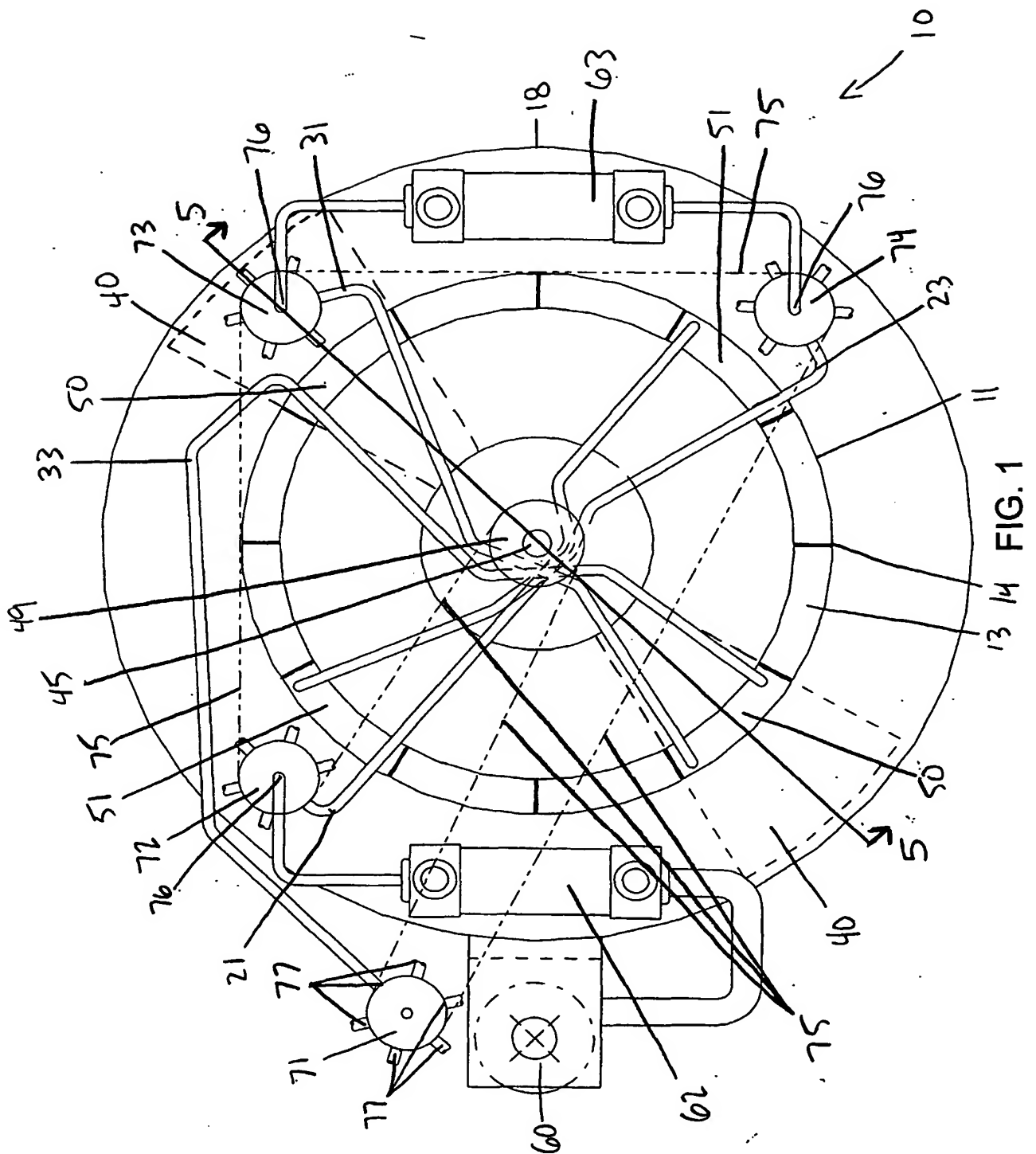
14 plurality of magnetic regenerator compartments, wherein at a second
15 rotational position the magnet passes a relatively low magnetic field
16 strength through the first magnetic regenerator compartment and passes
17 a relatively high magnetic field strength through the second magnetic
18 regenerator compartment; and

19 (c) a valve having an axial port, a rotating inner assembly,
20 and a plurality of radial ports, wherein the rotating inner assembly rotates
21 to connect fluid flow between the axial port and one or more of the radial
22 ports.

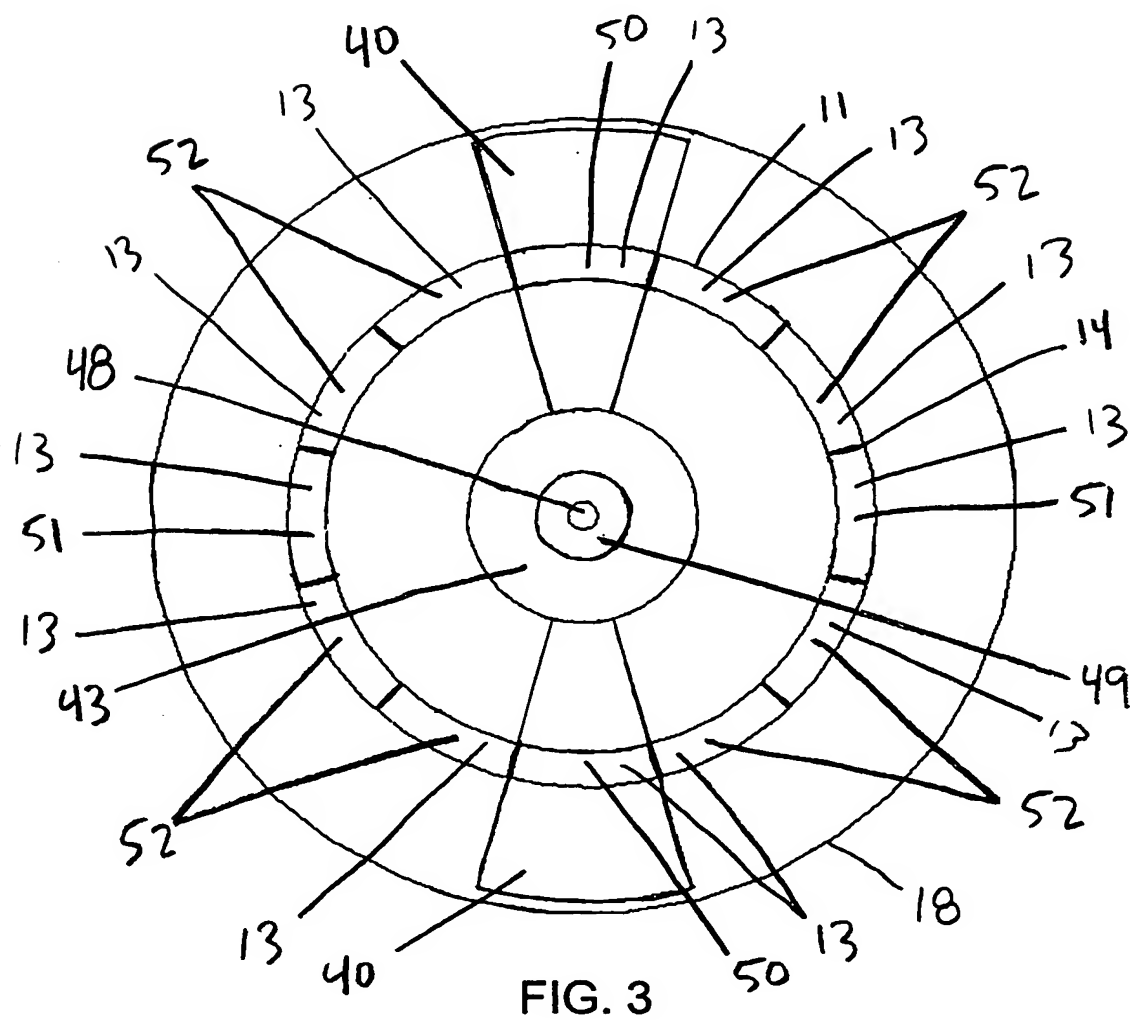
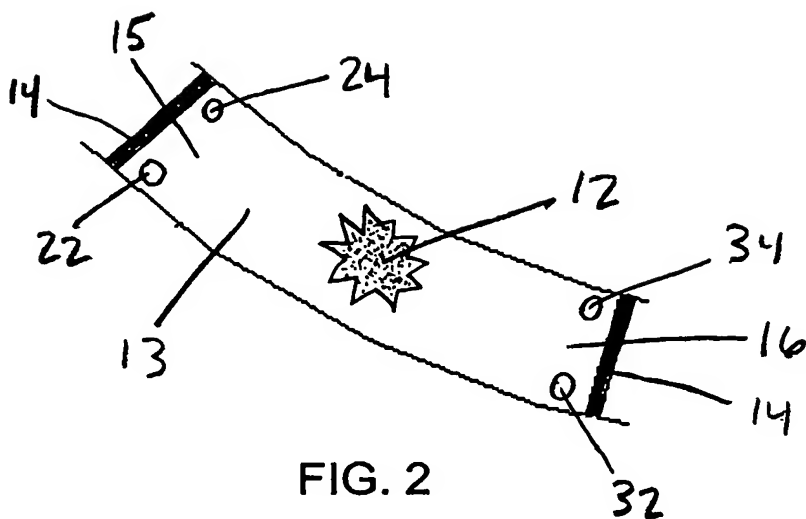
1 25. The refrigeration apparatus of Claim of 24 wherein the
2 rotating inner assembly rotates synchronously with the rotation of the
3 magnet.

1 26. The refrigeration apparatus of Claim of 24 wherein the
2 magnet cross section is C-shaped.

1/5



2/5



3/5

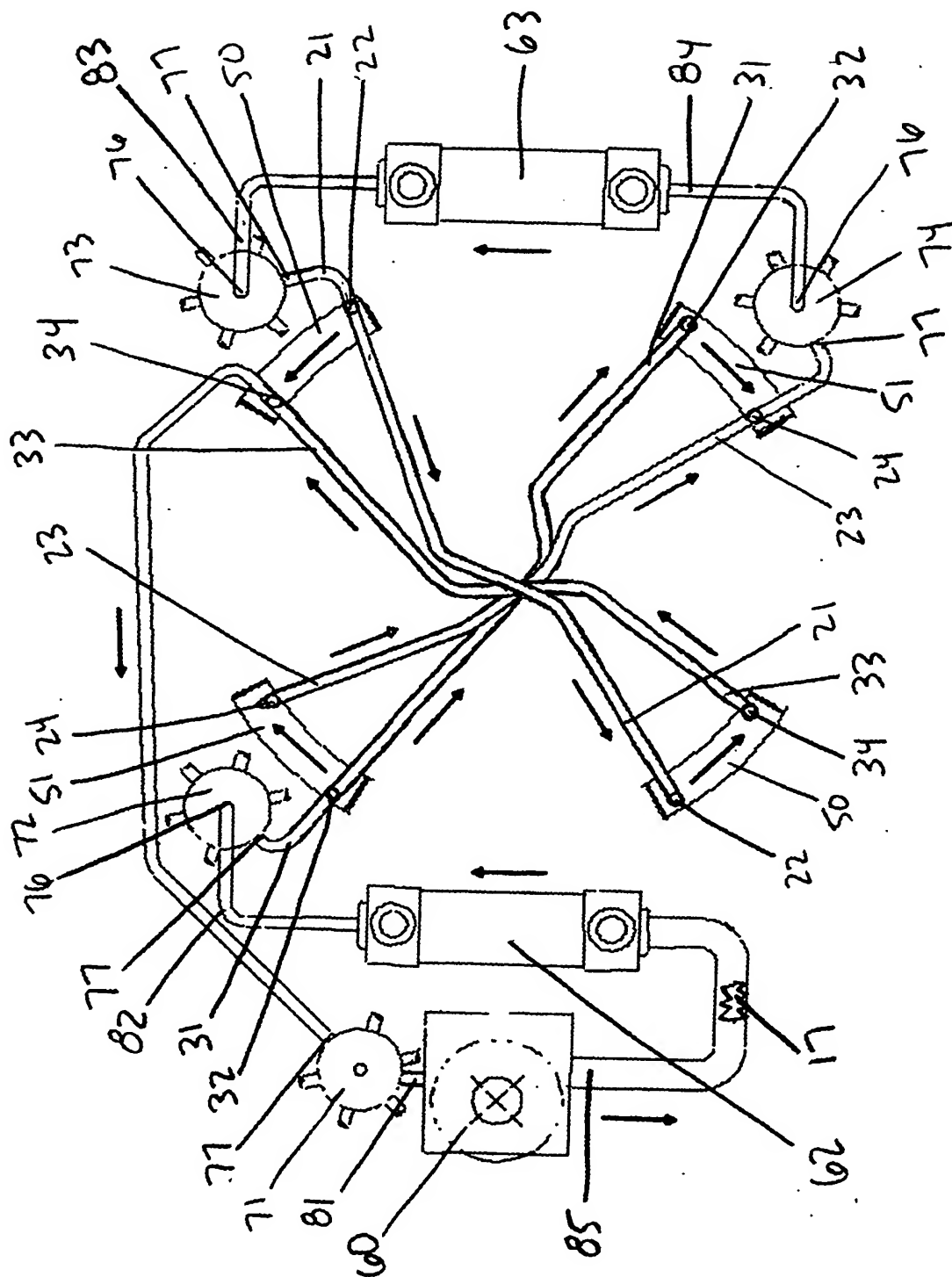


FIG. 4

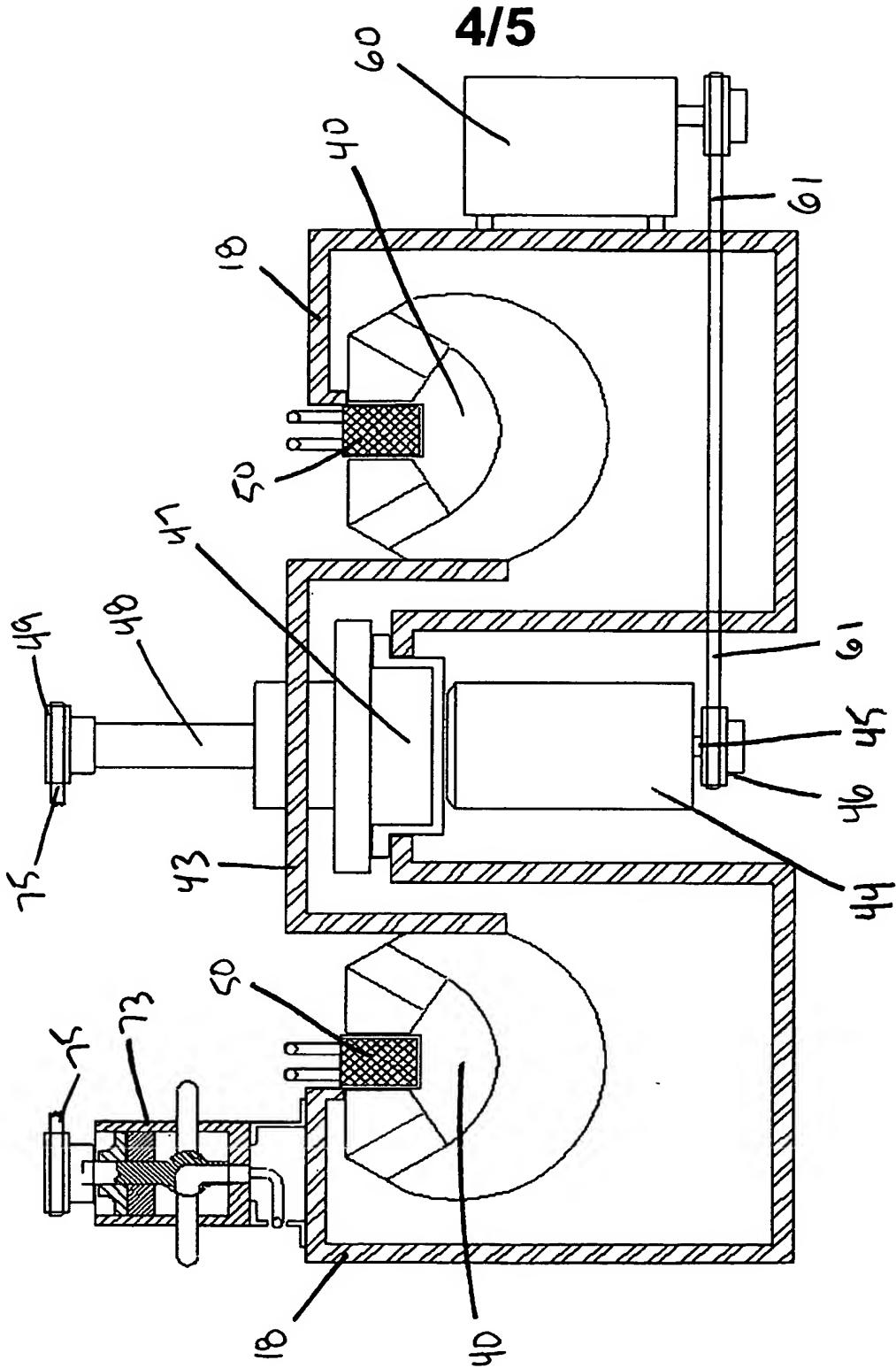


FIG. 5

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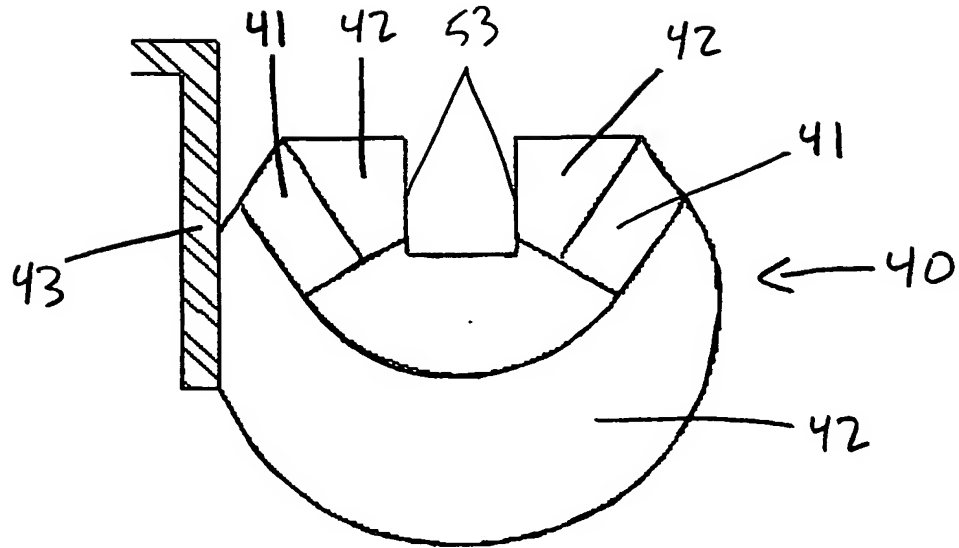


FIG. 6

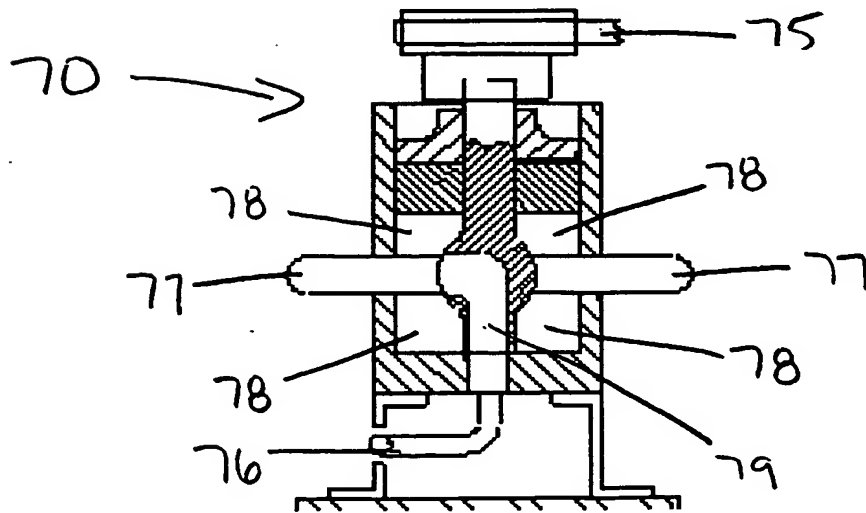


FIG. 7

INTERNATIONAL SEARCH REPORT

International application No.

PCT/US02/39656

A. CLASSIFICATION OF SUBJECT MATTER

IPC(7) : F25B 21/00
US CL : 62/ 3.1

According to International Patent Classification (IPC) or to both national classification and IPC

B. FIELDS SEARCHED

Minimum documentation searched (classification system followed by classification symbols)

U.S. : 62/ 3.1, 467

Documentation searched other than minimum documentation to the extent that such documents are included in the fields searched
None

Electronic data base consulted during the international search (name of data base and, where practicable, search terms used)
None

C. DOCUMENTS CONSIDERED TO BE RELEVANT

Category *	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
Y	US 4,408,463 A (BARCLAY) 11 October 1983 (11.10.1983), Figures 5-7; column 4, line 16 - column 7, line 68.	1-7
Y	US 5,091,361 A (HED) 25 February 1992 (25.02.1992), figures 14 and 15; column 16, lines 63-66.	1-7
A	US 4,727,721 A (PESHKA et al) 01 March 1988 (01.03.1988), see entire document.	1-26
A	US 5,249,424 A (DEGREGORIA et al) 05 October 1993 (05.10.1993), see figures 12-16.	1-26
A	US 4,507,927 A (BARCLAY) 02 April 1985 (02.04.1985), see entire document.	1-26
A	US 4,727,722 A (KIROL) 01 March 1988 (01.03.1988), see entire document.	1-26
A,P	US 6,467,274 B2 (BARCLAY et al) 22 October 2002 (22.10.2002), see figure 2.	1-26
A	US 5,444,983 A (HOWARD) 29 August 1995 (29.08.1995), see entire document.	1-26



Further documents are listed in the continuation of Box C.



See patent family annex.

* Special categories of cited documents:		*T	later document published after the international filing date or priority date and not in conflict with the application but cited to understand the principle or theory underlying the invention
"A"	document defining the general state of the art which is not considered to be of particular relevance	"X"	document of particular relevance; the claimed invention cannot be considered novel or cannot be considered to involve an inventive step when the document is taken alone
"E"	earlier application or patent published on or after the international filing date	"Y"	document of particular relevance; the claimed invention cannot be considered to involve an inventive step when the document is combined with one or more other such documents, such combination being obvious to a person skilled in the art
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"O"	document referring to an oral disclosure, use, exhibition or other means		
"P"	document published prior to the international filing date but later than the priority date claimed		

Date of the actual completion of the international search

28 February 2003 (28.02.2003)

Date of mailing of the international search report

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